

APPLICATION OF THE DEFORMATION FRACTURE CRITERION TO CRACKING OF DISC SPECIMENS WITH A CENTRAL NARROW SLOT

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Abstract: Using the method of singular integral equations, the elastic-plastic problem for cracked Brazilian disk was solved. Based on the Dugdale model and deformation fracture criterion, the relationships between critical load, notch tip opening displacement and length of the plastic strips were established. Also, the comparison between the present solution for the finite domain and the known solution obtained for the semi-infinite notch in the elastic plane was performed.

Key words: cracked Brazilian disc, Dugdale model, plastic strips, fracture deformation criterion, singular integral equations

1. INTRODUCTION

The application of the deformation fracture criterion in determining the basic fracture mechanics parameters requires the knowledge of the relationship between the load level and the opening displacement at the crack tip. This means that for an arbitrary test element, not only the stress field should be determined, but also the strain field considering the changes taking place in the fracture process zone. The general solutions for a crack or notch in the infinite plane are known ([1], see also the literature [2]) but for a particular specimen, these solutions can be only regarded as asymptotic. This work aims to determine the relationship between the load level and the opening displacement at the notch tip for a cylindrical specimen with a central narrow slot.

The Brazilian test is a simple indirect testing method, which is used to obtain the tensile strength of brittle materials such as concrete, rock, and rock-like materials. An up-to-date review of works concerning various aspects of the Brazilian test can be found in the literature [3,4].

A disc with an internal central crack as a convenient experimental specimen was considered analytically by [5–9]. These results concerning stress field distribution and values of the stress intensity factors were confirmed by Atkinson et al. [10] and Awaji and Sato [11]. Recent works devoted to the investigation of the fracture process in quasi-brittle materials using a compressed disc with a central slot are [12–21].

The basic material parameter in fracture mechanics is the critical stress intensity factor determined experimentally on specimens with initial cracks. In the case of metals, the procedure for determining this parameter is standardised and widely used. The fracture process in this case begins with the fatigue-initiated crack. For quasi-brittle materials such as concrete, ceramics or rocks, it is difficult to obtain an initial crack with strictly defined parameters. Usually, the initial crack is produced at the specimen forming stage. In this way, slots of quite significant (2–4 mm) width and rounded tips are obtained. Also, previously cited

sources devoted to analytical investigation of stress concentration in disc specimens refer to strict mathematical crack, i.e. the crack of zero width.

The method of determining the critical stress intensity factor of a quasi-brittle material on a compressed disc specimen with a centrally located narrow slot is presented. Knowing the value of the critical load and standard material constants, the critical stress intensity factor is calculated using the deformation fracture criterion [22] based on the Dugdale model [23,24]. The results were compared to previously published approximate relationships for semi-infinite U-notch in an elastic-plastic plate subjected to tension [25].

2. PROBLEM FORMULATION

There are two equal ($l_2 = l_3 = \ell$) linear cuts (contours L_2 and L_3) emanating from the slot vertices and placed on the axis Ox . These cuts will model the fracture process zone as plastic strips [26]. We assume that the Tresca-Saint Venant plasticity condition is fulfilled in these bands. The overall unknown relative extent of all defects weakening disc specimen we define as $\gamma = (\ell_0 + \ell)/R$.

The set of dimensionless geometric parameters ($\varepsilon_0, \gamma_0, \gamma$) fully describes the domain under consideration. Assuming radius R as the basic unit length, we obtain relationships

$$\ell_0 = \gamma_0 R, \quad \ell = (\gamma - \gamma_0) R, \quad \rho = \varepsilon_0 \gamma_0 R. \quad (1)$$

Suppose that the hole edge (the smooth contour L_1) is free of applied loads. The disc is loaded by two concentrated forces P , which compress the specimen along the Ox axis (Fig.1). Such type of loading causes the concentration of tensile stresses in vertices ($\pm \ell_0$) of the hole.

The problem will be solved using the singular integral equation method [27] (see also Savruk and Kazberuk [2]). Complex stress potentials are written in the form [27]

$$\Phi_*(z) = \Phi_0(z) + \Phi(z), \quad \Psi_*(z) = \Psi_0(z) + \Psi(z), \quad (2)$$

where functions [28]:

$$\Phi_0(z) = \sigma_p \frac{z^2 + R^2}{2(z^2 - R^2)}, \quad \Psi_0(z) = \sigma_p \frac{2R^4}{(z^2 - R^2)^2}. \quad (3)$$

Nominal stress $\sigma_p = P/(\pi R)$ is equal to normal stress σ_y alongside Ox axis, $z = x + iy$.

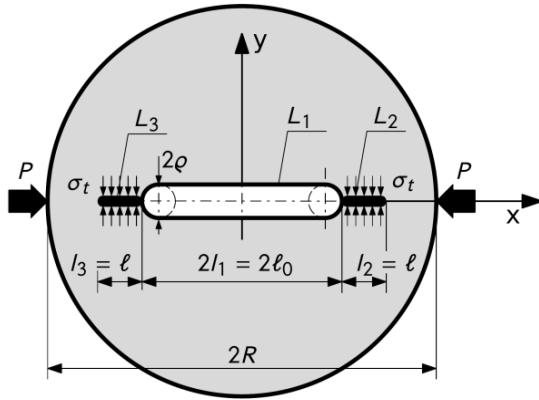


Fig. 1. Disc specimen weakened by central narrow slot and plastic strips subjected to compression by concentrated forces

Functions $\Phi_0(z)$, $\Psi_0(z)$ describe the stress state in the solid disc (i.e. without a hole) loaded by concentrated forces, whereas potentials $\Phi(z)$, $\Psi(z)$ characterise the disturbed stress state induced by the opening (L_1) and cuts (L_2 , L_3). These potentials are written in the following form [27]:

$$\Phi(z) = \frac{1}{2\pi} \int_L \left\{ \left[\frac{1}{t-z} + \frac{\bar{t}}{z\bar{t}-R^2} \right] g'(t) dt + \frac{z(t\bar{t}-R^2)(z\bar{t}-2R^2)}{R^2(z\bar{t}-R^2)^2} \overline{g'(t)} d\bar{t} \right\}, \quad (4)$$

$$\Psi(z) = \frac{1}{2\pi} \int_L \left\{ \left[\frac{\bar{t}^3}{(z\bar{t}-R^2)^2} - \frac{\bar{t}}{(t-z)^2} \right] g'(t) dt + \left[\frac{1}{t-z} + \frac{\bar{t}}{z\bar{t}-R^2} + \frac{\bar{t}(z\bar{t}-3R^2)(t\bar{t}-R^2)}{(z\bar{t}-R^2)^3} \right] \overline{g'(t)} d\bar{t} \right\}.$$

Here $g'(t)$ ($t \in L_k$, $k = 1,2,3$) is an unknown function of the derivative of displacement discontinuity vector across the cut contour.

The boundary condition at the contour L has the following form:

$$N(t) + iT(t) = p(t) \quad t \in L, \quad L = \bigcup_{k=1}^3 L_k, \quad (5)$$

where N and T are normal and tangential components of the stress vector. The right side of the Eq. (5) is equal [27]:

$$p(t) = \sigma_k - \left\{ \Phi_0(t) + \overline{\Phi_0(t)} + \frac{dt}{dt} \left[t\overline{\Phi_0'(t)} + \overline{\Psi_0(t)} \right] \right\}, \quad t \in L, \quad (6)$$

where

$$\sigma_k = \begin{cases} 0, & k = 1, \\ \sigma_t, & k = 2,3 \end{cases} \quad (7)$$

In further calculations, it was assumed that σ_t is equal to the material strength of the specimen determined in the Brazilian test (compressed disc without slot).

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Fulfilling boundary condition Eq. (5) using potentials Eq. (4) we obtain the system of singular integral equations with unknown functions $g'_m(t)$ ($m = 1,2,3$)

$$\frac{1}{\pi} \sum_{k=1}^3 \int_{L_k} \left[K_{km}(t,t') g'_m(t) dt + L_{km}(t,t') \overline{g'_m(t)} d\bar{t} \right] = p_m(t'), \quad (8)$$

$$t' \in L_m, \quad m = 1,2,3,$$

where kernels are as follows:

$$K(t,t') = f_1(t,t') + \overline{f_2(t,t')} + \frac{dt'}{dt} \left[t' \overline{g_2(t,t')} + \overline{h_2(t,t')} \right],$$

$$L(t,t') = f_2(t,t') + \overline{f_1(t,t')} + \frac{dt'}{dt} \left[t' g_1(t,t') + \overline{h_1(t,t')} \right],$$

and

$$f_1(t,t') = \frac{1}{2} \left[\frac{1}{t-t'} + \frac{\bar{t}}{t'\bar{t}-R^2} \right],$$

$$f_2(t,t') = \frac{t'(t\bar{t}-R^2)(t'\bar{t}-2R^2)}{2R^2(t'\bar{t}-R^2)^2},$$

$$g_1(t,t') = \frac{1}{2} \left[\frac{1}{(t-t')^2} - \frac{\bar{t}^2}{(t'\bar{t}-R^2)^2} \right],$$

$$g_2(t,t') = \frac{R^2(t\bar{t}-R^2)}{(t'\bar{t}-R^2)^3},$$

$$h_1(t,t') = \frac{1}{2} \left[-\frac{\bar{t}}{(t-t')^2} + \frac{\bar{t}^3}{(t'\bar{t}-R^2)^2} \right],$$

$$h_2(t,t') = \frac{1}{2} \left\{ \frac{1}{t-t'} + \frac{\bar{t} [4R^4 - 3R^2\bar{t}(t+t') + t'\bar{t}^2(t+t')]}{(t'\bar{t}-R^2)^3} \right\}.$$

3. NUMERICAL SOLUTION OF INTEGRAL SINGULAR EQUATIONS

We assume a clockwise direction of tracing the contour L_1 so the elastic region stays on the left during tracing. Taking into consideration the symmetry of the contour concerning both coordinate axes, we can write its parametric equation in the form [2]:

$$t = R\omega_1(\xi) = \begin{cases} \omega_q(\xi), & 0 \leq \xi < \pi/2, \\ -\overline{\omega_q(\pi - \xi)}, & \pi/2 \leq \xi < \pi, \\ -\omega_q(\xi - \pi), & \pi \leq \xi < 3\pi/2, \\ \overline{\omega_q(2\pi - \xi)}, & 3\pi/2 \leq \xi < 2\pi. \end{cases} \quad (9)$$

Here, the function $\omega_q(\xi)$ describes the segment of contour L_1 laying in the fourth quarter of the coordinate system:

$$\omega_q(\xi) = \begin{cases} 1 - \varepsilon_0 + \varepsilon_0(\cos c\xi - i \sin c\xi), & 0 \leq \xi < \pi/(2c), \\ \varepsilon_0 c(\pi/2 - \xi) - i\varepsilon_0, & \pi/(2c) \leq \xi \leq \pi/2, \end{cases} \quad (10)$$

where parameter $c = 1 + 2(1/\varepsilon_0 - 1)/\pi$. Total curve L_1 length equals to $2\pi\varepsilon_0\gamma_0 Rc$.

The parametric equation describing cut L_2 was written in the form

$$t = R\omega_2(\xi) = R \left[\gamma_0 + \frac{1}{2}(\gamma - \gamma_0)(1 + \xi) \right], \quad -1 \leq \xi \leq 1 \quad (11)$$

Contour L_3 is symmetrical to L_2 concerning the Oy axis so

$$t = R\omega_3(\xi) = -R\omega_2(\xi), \quad -1 \leq \xi \leq 1. \quad (12)$$

Introducing substitutions

$$\begin{aligned}
 t &= R \omega_1(\xi), \quad t' = R \omega_1(\eta), \\
 t, t' &\in L_1, \quad 0 \leq \xi, \eta \leq 2\pi, \\
 t &= R \omega_k(\xi), \quad t' = R \omega_k(\eta), \\
 t, t' &\in L_k, \quad k = 2,3, \quad -1 \leq \xi, \eta \leq 1,
 \end{aligned} \tag{13}$$

we reduce the system of integral equations Eq. (8) to the canonical form

$$\begin{aligned}
 \frac{1}{\pi} \int_0^{2\pi} [M_{1m}(\xi, \eta) g'_1(\xi) + N_{1m}(\xi, \eta) \overline{g'_1(\xi)}] d\xi + \\
 + \frac{1}{\pi} \sum_{k=2-1}^3 \int_{-1}^1 [M_{km}(\xi, \eta) g'_k(\xi) + N_{km}(\xi, \eta) \overline{g'_k(\xi)}] d\xi = p_m(\eta),
 \end{aligned} \tag{14}$$

$m = 1, 2, 3,$

where

$$\begin{aligned}
 M_{km}(\xi, \eta) &= RK_{km}^*(R\omega_k(\xi), R\omega_m(\eta)), \\
 N_{km}(\xi, \eta) &= RL_{km}^*(R\omega_k(\xi), R\omega_m(\eta)), \\
 g'_k(\xi) &= g'(R\omega_k(\xi))\omega'_k(\xi), \\
 p_m(\eta) &= p(R\omega_m(\eta)).
 \end{aligned}$$

The solution of the system of integral equations (14) consists of three complex functions $g'_k(\xi)$ assigned to the contours L_k . Function $g_1(\xi)$ ($0 \leq \xi \leq 2\pi$) is 2π -periodic continuous function. However, in order to obtain a sufficiently accurate numerical solution, we have to densify quadrature nodes and collocation points in the vicinity of narrow slot tips. We use here a variant of sigmoid transformation [29,30] adapted to periodic case [29]:

$$\xi = G(\tau) = \tau - \frac{1}{2} \sin 2\tau, \quad 0 \leq \tau \leq 2\pi. \tag{15}$$

Consequently, the function we are looking for is as follows

$$u_1(\tau) = g'_1(G(\tau)), \quad 0 \leq \tau \leq 2\pi. \tag{16}$$

A solution of the system of integral equations Eq. (14) for contours L_2 and L_3 is sought in the class of functions, which have an integrable singularity at the ends of the integration interval

$$g'_k(\xi) = \frac{u_k(\xi)}{\sqrt{1-\xi^2}}, \quad -1 \leq \xi \leq 1, \tag{17}$$

where $u_k(\xi)$ ($k = 2,3$) are continuous functions.

Finally, a modified system of the singular integral equations Eq. (14) takes the form

$$\begin{aligned}
 \frac{1}{\pi} \int_0^{2\pi} [M_{1m}(\xi, \eta) u_1(\tau) + N_{1m}(\xi, \eta) \overline{u_1(\tau)}] G(\tau) d\tau + \\
 + \frac{1}{\pi} \sum_{k=2-1}^3 \int_{-1}^1 [M_{km}(\xi, \eta) u_k(\xi) + N_{km}(\xi, \eta) \overline{u_k(\xi)}] d\xi = p_m(\eta),
 \end{aligned} \tag{18}$$

$m = 1, 2, 3,$

In points $t = \pm \ell_0$ where contours L_2 and L_3 intersect contour L_1 the values of $g'_k(-1)$ ($k = 2,3$) must be finite, thus we should provide two additional equations

$$u_k(-1) = 0, \quad k = 2,3. \tag{19}$$

For numerical integration of the singular integral equation (18) two different methods must be used. For the closed-loop contour L_1 , we apply the midpoint rule [31] and Gauss-Chebyshev quadrature [27] for L_2 and L_3 contours. Finally we get a system of

complex linear algebraic equations which is the discrete analogue of the respective system of integral equations (18)

$$\begin{aligned}
 \frac{2}{n_1} \sum_{i=1}^{n_1} [M_{1m}(\xi_i, \eta_j) u_1(\tau_i) + N_{1m}(\xi_i, \eta_j) \overline{u_1(\tau_i)}] G(\tau_i) + \\
 + \sum_{k=2}^3 \left\{ \frac{1}{n_k} \sum_{i=1}^{n_k} [M_{km}(\xi_i, \eta_j) u_k(\xi_i) + N_{km}(\xi_i, \eta_j) \overline{u_k(\xi_i)}] \right\} = p_m(\eta_j),
 \end{aligned} \tag{20}$$

$$m = 1, \quad j = 1, \dots, n_k,$$

$$m = 2, 3, \quad j = 1, \dots, (n_k - 1),$$

where quadrature nodes and collocation points are determined by formulas:

$$\begin{aligned}
 \xi_i &= G(\tau_i), \quad \tau_i = \frac{\pi(2i-1)}{n_1}, \quad i = 1, \dots, n_1, \\
 \eta_j &= G(\theta_j), \quad \theta_j = \frac{2\pi(j-1)}{n_1}, \quad j = 1, \dots, n_1, \\
 \xi_i &= \cos \frac{\pi(2i-1)}{2n_k}, \quad i = 1, \dots, n_k, \quad k = 2,3, \\
 \eta_j &= \cos \frac{\pi j}{n_k}, \quad j = 1, \dots, (n_k - 1), \quad k = 2,3.
 \end{aligned} \tag{21}$$

The linear system Eq. (20) consists of $n_1 + (n_2 - 1) + (n_3 - 1)$ complex equations. Using Lagrange interpolation on Chebyshev nodes [27] to conditions Eq. (19), we obtain two missing equations

$$\frac{1}{n_k} \sum_{i=1}^{n_k} (-1)^{i+n_k} \tan \frac{\pi(2i-1)}{4n_k} u_k(\xi_i) = 0, \quad k = 2,3. \tag{22}$$

The right side of the Eq. (20) can be easily calculated using the relationship Eq. (6). Introducing the relationship $\lambda = \sigma_p / \sigma_t$ ($\sigma_p = P / (\pi R)$) as a relative load level parameter, we can write down $p_m(\eta_j)$ in compact form:

$$p_m(\eta_j) = \begin{cases} p_1(\eta_j), & m = 1, \\ \left(1 - \frac{1}{\lambda}\right) p_1(\eta_j), & m = 2,3, \end{cases} \tag{23}$$

where

$$p_1(\eta_j) = \sigma_p \frac{|\omega_k(\eta_j)|^2 - 1}{\omega_k(\eta_j)^2 - 1} \left[\frac{2 \overline{\omega'_k(\eta_j)}}{\omega_k(\eta_j)^2 - 1} - \frac{|\omega_k(\eta_j)|^2 + 1}{\omega_k(\eta_j)^2 - 1} \right], \tag{24}$$

$k = 1, 2, 3.$

The solution to the problem is symmetrical concerning the axis Ox i Oy . The conditions resulting from symmetry concerning the sought function $u_k(\xi)$ and necessary kernel modifications are described in detail in Savruk et al. [32] (see also the literature [2]). Thus, the rank of the linear system Eq. (20) and (23) can be easily reduced by a factor of four.

The obtained values of sought function $u(\xi_k)$ fully determine the stress-strain state in the whole elastic region through an integral representation of complex stress potentials Eq. (4).

The slot edge (contour L_1) is free of applied loads, then the contour stress at the edge can be calculated using a simple formula [2].

$$\sigma_s = -4\sigma_p \Im \frac{u_1(\xi)}{\omega'_1(\xi)} = -4\sigma_p \Im \frac{u_1(\tau)}{\omega'_1(G(\tau))}. \tag{25}$$

Stress intensity factors in crack tips K_I and K_{II} can be directly expressed through the sought function $g'_k(t)$ Eq. (17). Let us introduce corresponding dimensionless stress intensity factors F_I and F_{II} using the following relationship

$$K_I^+ - iK_{II}^+ = (F_I^+ - iF_{II}^+) \sigma_p \sqrt{\pi R}. \tag{26}$$

Here upper indexes (+) indicate crack tip at $\xi = +1$. Taking into account relation Eq. (17), we get coefficients F_I and F_{II} [27]

$$F_I^+ - iF_{II}^+ = -\sqrt{|\omega_{k'}(+1)|} \frac{u_k(+1)}{\omega_{k'}(+1)}, \quad k = 2,3, \quad (27)$$

where

$$u_k(+1) = -\frac{1}{n} \sum_{i=1}^{n_k} (-1)^i u_k(\xi_i) \cot \frac{\pi(2i-1)}{4n}, \quad k = 2,3. \quad (28)$$

Cracks L_2 and L_3 simulate fracture process zones (plastic strips) at the tips of narrow slot L_1 , thus stresses at the crack L_k end must be finite

$$g'_k(t = \pm(\ell_0 + \ell)) = g'_k(R\omega_k(+1)) = 0, \quad \rightarrow$$

$$u_k(+1) = 0, \quad k = 2,3. \quad (29)$$

This condition allows us to calculate the unknown length $\ell = (\gamma - \gamma_0)R$ using a simple iteration process.

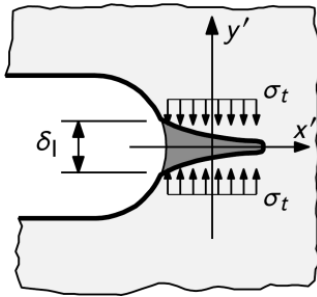


Fig. 2. U-notch tip opening displacement

The opening displacement in the notch tip (Fig. 2) can be calculated based on the known [27] relationship between function $g_k(t)$ ($k = 2,3$) and displacement discontinuity ($v_k^+ - v_k^-$) across the contour L_k

$$2G \frac{d}{dx'} (v_k^+ - v_k^-) = (1 + \kappa) g'_k(x'),$$

$$x' \in L_k, \quad k = 2,3, \quad (30)$$

where x' is a local abscissa at contour L_k , G – shear modulus, κ – Muskhelishvili's constant.

In plane stress state $(1 + \kappa)/(4G) = 2/E$, so we come to the formula for crack opening displacement in its left tip $x' = l_k^-$ ($x = \pm \ell_0$), i.e. in slot tips, in the form [32]

$$\delta_1 = \delta_1(l_k^-) = \frac{4}{E} \Re g_k(l_k^-) = -\frac{4}{E} \Re \int_{l_k^-}^{l_k^+} g'_k(t) dt =$$

$$= -\frac{4R\sigma_p}{E} \Re \int_{-1}^{+1} \frac{u_k(\xi)}{\sqrt{1-\xi^2}} d\xi =$$

$$= -\frac{\sigma_p R}{E} \frac{4\pi}{n_k} \sum_{i=1}^{n_k} \Re u_k(\xi_i), \quad k = 2,3. \quad (31)$$

4. NUMRICAL RESULTS

The calculations were performed for the constant rounding radius of slot tips $\rho_0 = 1/75 R$. Relative slot span γ_0 was chosen from the set $\gamma_0 = \{0.1, 0.2, 0.3, 0.4, 0.5, 0.6\}$, so the relative rounding radius of the slot vertices can be easily calculated as

$\varepsilon_0 = 1/(75\gamma_0)$. For every slot geometry and for arbitrary load level $\lambda = \sigma_p/\sigma_t$ ($\lambda_{\min} < \lambda < 1$) the plastic strips range $\gamma = (\ell_0 + \ell)/R$ and notch tip opening displacement δ_1 were calculated. These values are presented in Figs 3 and 4 respectively.

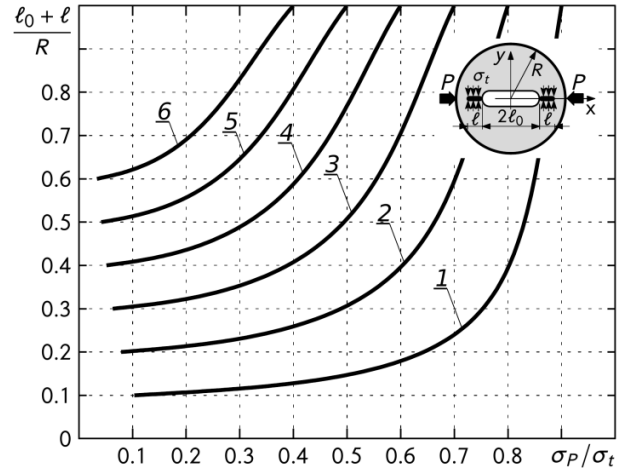


Fig. 3. Relative plastic strips range versus relative load level for the following relative slot span: 1 – 0.1, 2 – 0.2, 3 – 0.3, 4 – 0.4, 5 – 0.5, 6 – 0.6

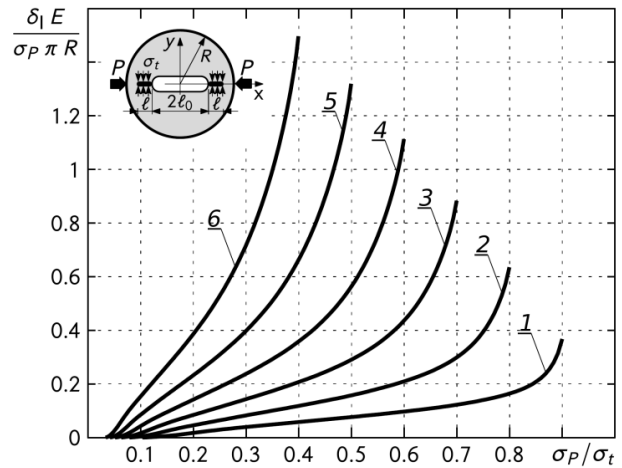


Fig. 4. Dimensionless notch tip opening displacement versus relative load level for the following relative slot span: 1 – 0.1, 2 – 0.2, 3 – 0.3, 4 – 0.4, 5 – 0.5, 6 – 0.6

As it can be easily seen, the values of relative load level λ start with a certain value λ_{\min} . Plastic strips arise when the maximum normal stress σ_{\max} at the vertex of the narrow slot reaches the limit value σ_t . If we denote the stress concentration factor in the rounded notch tip as k , then $k\sigma_p = \sigma_{\max} \leq \sigma_t$ and the minimum value of the load level parameter will be equal to $\lambda_{\min} = 1/k$. Stress concentration factor can be easily calculated (see Savruk and Kazberuk [2]) by solving the system of equations (20) taking into account contour L_1 only and then using Eq. (26). A list of the stress concentration factors for all γ_0 values used here is shown in Tab. 1.

When the plastic deformations are small, the critical stress intensity factor K_c can be calculated with the formula [1,33]

$$K_c = \sqrt{\delta_1 E \sigma_t} = \sqrt{\tilde{\delta}_1 \lambda} \sigma_t \sqrt{\pi R}. \quad (32)$$

Tab. 1. Stress concentrations factors at the rounded vertex of narrow slot and dimensionless stress intensity factors at the tips of a corresponding central crack in disc specimen

$\gamma_0 = \frac{\ell_0}{R}$	$\varepsilon_0 = \frac{\rho}{\ell_0}$	$k = \frac{\sigma_{max}}{\sigma_p}$	$F_I = \frac{K_I}{\sigma_p \sqrt{\pi R}}$
0.1	0.1333	9.685	0.3210
0.2	0.0667	12.75	0.4741
0.3	0.0444	15.89	0.6220
0.4	0.0333	19.54	0.7866
0.5	0.0267	24.04	0.9838
0.6	0.0222	29.90	1.2364

Let's compare the obtained results with the approximate estimation calculated for a semi-infinite U-notch [25]. The relative load level parameter is equal

$$\gamma^\infty = \frac{1}{\sqrt{2\pi\rho}} \frac{K_I}{\sigma_t} = \frac{F_I}{\sqrt{2\varepsilon_0\gamma_0}} \frac{1}{\sigma_t} \frac{P}{\pi R} = \frac{F_I}{\sqrt{2\varepsilon_0\gamma_0}} \lambda, \quad (33)$$

Dimensionless notch opening displacement is equal [25]

$$\sqrt{\delta_1^\infty} = 1 - \frac{1}{R_1^2 (\gamma^\infty)^2}, \quad (34)$$

where the stress rounding factor [34] is equal $R_1 = 2.993$ [35].

The value of the critical stress intensity factor, calculated based on the criteria condition Eq. (32), is equal

$$K_c^\infty = \sqrt{\delta_1^\infty} K_I = F_I \lambda \sigma_t \sqrt{\delta_1^\infty} \pi R, \quad (35)$$

where dimensionless stress intensity factors F_I are shown in Tab 1.

In Fig. 5. the comparison of the values of the critical stress intensity factors calculated for the notched disk Eq. (33) and for semi-finite U-notch in the tensile plane Eq. (36) is shown. It can be seen that coarse approximation is only valid for the smallest slot ($\gamma_0 = 0.1$) under load level $0.4 < \lambda < 0.8$. For relative slot span $\gamma_0 = 0.2$ differences between exact and approximated values are much greater – nearly 30% at $\lambda \sim 0.5$.

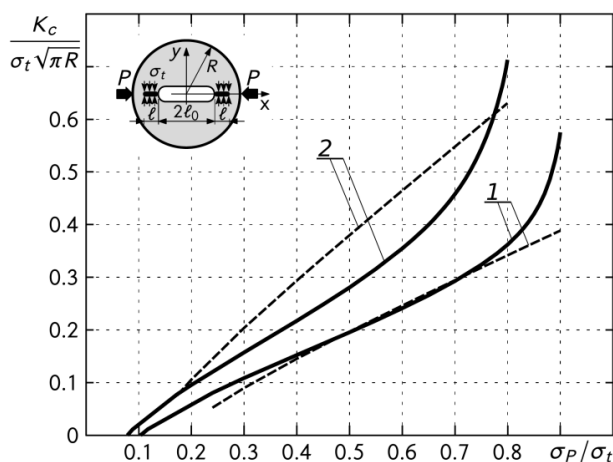


Fig. 5. Comparison of the values of the critical stress intensity factors calculated for notched disc (solid lines) and at the vertex of the semi-infinite U-notch (dashed lines) as a function of the relative load level for the following parameters: $1 - \gamma_0 = 0.1, 2 - \gamma_0 = 0.2$

5. CONCLUSIONS

The elastic-plastic problem for the Brazilian disc with a central narrow slot in the plane stress state condition was solved. The solution was obtained by the method of singular integral equations using complex stress potentials for a system of cracks and openings in the two-dimensional circular elastic domain. All necessary analytical background was documented in detail. Based on the Dugdale model of fracture process zone and deformation fracture criterion, the relationships between critical load, notch tip opening displacement and length of the plastic strips were established. Numerical calculations for arbitrary but representative sets of geometrical parameters were performed. The correctness of the solution was checked by comparing the results with the approximate relationships obtained for the problem of semi-infinite U-notch in an elastic plane subjected to unidirectional tension. For a small relative slot span, both results are in good agreement.

The presented approach, despite the obvious simplifications resulting from the adopted assumptions (plane stress state, fracture process zone as a plastic strip), can be used to estimate the fracture mechanics parameters of quasi-brittle materials determined in the Brazilian test.

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